

1. A 10 m long beam, simply supported at its ends, carries udl of 10 kN/m over the span. The maximum deflection in terms of EI is :->1302/EI
2. A 4 m long beam, simply supported at its ends, carries a point load 'W' at its centre. If the slope at the ends of the beam is 10, then the deflection at the centre of the beam will be in 'mm' :->23.27
3. A 6 m long beam, simply supported at its ends, carries a point load 10 kN at its centre. The maximum deflection in terms of EI is :->45/EI
4. A beam is a :->one dimensional structure
5. A beam is said to be of uniform strength, if :->bending stress is same at every section along its longitudinal **axis**
6. A beam is supported with equal over hangs over the two supports. Each overhang is 'a' and supported span is 'L'. the deflection of the mid point of the beam due to two point load 'W' each acting on the two ends is :-
7. A beam of overall length l rests on two simple supports with equal overhangs on both sides. Two equal loads act at the free ends. If the deflection at the centre of the beam is the same as at either end, then the length of >either overhang is :->0.152 l
8. A boiler shell 100 cm dia and is subjected to an internal pressure of 1.2 M N/m² and the hoop stress is 90 N/m². The plate thickness is :->12 mm
9. A boiler shell 100 cm dia and plate thickness 12 mm is subjected to an internal pressure of 1.2 M N/m². The hoop stress will be :->90 N/m²
10. A boiler shell is to be made of 20 mm thick plate having a limiting tensile stress of 125 N/mm². If the efficiencies of the longitudinal and circumferential joints are 80 % and 30 %, determine the maximum permissible dia of the shell for an internal pressure of 2.5N/mm² :->120 cm
11. A boiler shell is to be made of 20 mm thick plate having a limiting tensile stress of 125 N/mm². If the efficiencies of the longitudinal and circumferential joints are 80 % and 30 %, determine the maximum permissible dia of the shell for an internal pressure of 2.5N/mm² :->1. 875N/ mm²
12. A cantilever AB is subjected to a concentrated load at the free end the slope and deflection at the free end are and. If the same load is applied at mid span point, the deflection at the free end will be :- >
13. A cantilever beam of rectangular c/s carries a point load 'W' at its free end. If the depth of the beam is doubled and the load halved. The deflection at the free end, as compared to its original value, will be :- >1/4
14. A cantilever beam of span 2m is subjected to u.d.l of 10kN/m. If kN.mm². the maximum deflection in 'mm' is :->1000
15. A cantilever carries a u.d.l. (W) over its span and a force W act at its free end upward. The net deflection of the free end will be :->**upward**
16. A cantilever is subjected to a concentrated load W at the mid-point of the span. The slope at the free end will be :->
17. A cantilever is subjected to moment M at the free end. The deflection at free end is :- >
18. A compound cylinder is formed by :->shrinking one cylinder on to another
19. A continuous beam is one which is :->fixed at both ends
20. A cylinder can be assumed as a thin cylinder when the dia to thickness ratio is :->>20
21. A cylinder is to be designed as a thick cylinder when d/t ratio is :-><20
22. A cylinder of internal dia 0.60 m contains air at a pressure of 7.5N/mm². If the maximum permissible stress induced in the material is 758N/mm², find the thickness of the cylinder :->3 cm
23. A cylinder of internal dia 3m and of thickness 6 cm contains a gas. If the tensile stress in the material is not to exceed 70N/mm², determine the internal pressure of the gas :->2.8N/mm²
24. A cylindrical pipe of diameter 2.0m and thickness 2.0cm is subjected to an internal fluid pressure of 1.5N/mm². The longitudinal stress is :->37.5N/ mm²

25. A cylindrical pipe of diameter 2.0m and thickness 2.0cm is subjected to an internal fluid pressure of 1.5N/mm².
26. A cylindrical section having no joints is known as :->seamless section
27. A cylindrical shell made of mild steel plate of 100 cm diameter is to be subjected to an internal pressure of 10 kg/cm². If the material yields at 20 00 kg/cm², assuming factor of safety as four and using maximum principal stress theory, thickness of the plate will be :->10 mm
28. A cylindrical vessel whose ends are closed by means of rigid flange plates, is made of steel plate 4 mm thick. The length and internal dia of the vessel are 100 cm and 30 cm. Determine increase in length :->0.075 cm
29. A cylindrical vessel whose ends are closed by means of rigid flange plates, is made of steel plate 4 mm thick. The length and internal dia of the vessel are 100 cm and 30 cm. Determine increase in diameter. :->0.0095 cm
30. A cylindrical vessel whose ends are closed by means of rigid flange plates, is made of steel plate 4 mm thick. The length and internal dia of the vessel are 100 cm and 30 cm. Determine the longitudinal stress due to an internal pressure of 2 N/mm² :->37.5N/ mm²
The longitudinal stress is :->75N/ mm²
31. A cylindrical vessel whose ends are closed by means of rigid flange plates, is made of steel plate 4 mm thick. The length and internal dia of the vessel are 100 cm and 30 cm. Determine the hoop stress due to an internal pressure of 2 N/mm² :->75N/mm²
32. A frame is completely analysed when :->the variation in direct stress, shear and moment is found **throughout the frame**
33. A joint of a frame is subjected to three tensile force P, Q and R equally inclined to each other. If P = 10 tonnes, the other forces will be such that :->Q and R each is equal to 10 tonnes
34. A joint of a frame is subjected to three tensile forces A, B and C equally inclined to each other. If A = 10 kN, the other forces will be :->B=10 kN, C=10 kN
35. A member is being balanced at its end by two inclined members carrying equal force. For equilibrium the angle between the inclined bars must be :->120°
36. A one dimensional structure is one which :->one dimension is much larger than the other two
37. A pin jointed frame with number of members 'm' and number of joints 'j' will be a perfect frame, if :-> $m = 2j - 3$
38. A pin jointed plane frame with 'j' number of joints and 'n' number of members will be internally redundant, if :-> $n > (2j - 3)$
39. A plane structure is a structure :->the various members of which lie in a plane
40. A prismatic bar when subjected to pure bending assumes the shape of :->arc of a circle
41. A seamless water main 1 m in dia is required to carry water under a head of 100 m. If the maximum permissible tensile stress in the metal is 500 kg/cm², the thickness of metal required would be :->10 mm
42. A simply supported beam 'A' of length l, breadth b and depth d carries point load 'W'. Another beam 'B' has the same length and depth but its breadth is doubled. The deflection of beam 'B' will beas compared to beam 'A' :->1/2
43. A simply supported beam is subjected to a central concentrated load 'W'. The slope at the two ends is given by :->
44. A simply supported beam of circular c/s with diameter 'd' and length 'l' carries a concentrated load 'W' at the centre of the beam. The strength of the beam is proportional to :->d/

45. A simply supported beam of length l carries a load varying uniformly from zero at left end to maximum at right end. The maximum bending moment occurs at a distance of $l/3$ from left end
46. A simply supported beam of length L carries two equal unlike couples M at two ends. If the $EI =$ constant, the central deflection is $-\frac{2ML^2}{EI}$
47. A simply supported beam of span L carrying a u.d.l. registers a deflection of y cm at the centre. If the span of the beam is doubled, the deflection at the centre for the same u.d.l. would be $16y$
48. A simply supported beam of span L carrying a u.d.l. registers a deflection of y cm at the centre. If the u.d.l. is doubled, the deflection at the centre for the same span would be $2y$
49. A simply supported beam which carries a udl load over the whole span is propped at the centre of the span so that the beam is held to the level of the end supports, the reaction of the prop will be equal to $\frac{5}{8}$ the distributed load
50. A spherical pressure vessel is made of thin magnesium plate 0.25 cm thick. The main diameter of the sphere is 600 cm and allowable stress in tension is 900 kg/cm². The safe internal gas pressure for the vessel would be >1.5 kg/cm²
51. A spherical shell of 1.5 m dia has 1 cm thick wall. Determine the pressure that can increase its volume by 10 cm³. Take $E = 2$ times 105 N/mm² and $1/m = 0.3$ $\rightarrow 0.437$ N/mm²
52. A spherical shell of 80 cm dia is subjected to an internal pressure of 2 N/mm². Find the thickness required for the shell, if the allowable stress is 100 N/mm². $\rightarrow 0.4$ cm
53. A spherical shell of 80 cm dia is subjected to an internal pressure of 2 N/mm². Find the change in volume, if the allowable stress is 100 N/mm². Take $E = 2$ times 105 N/mm² and $1/m = 0.3$ $\rightarrow 0.00105$
54. A statically determinate structure is the one which \rightarrow can be analysed with the equations of statics alone
55. A steel cylinder 20 m long having a mean diameter of 3 m and a wall thickness of 20 mm is to contain compressed air at a pressure of 4 MN/m². The poisson's ratio is 0.3 and Young's modulus of cylindrical material is 210GN/m². The longitudinal stress in the cylinder material will be $\rightarrow 120$ MN/m
56. A steel cylinder 20 m long having a mean diameter of 3 m and a wall thickness of 20 mm is to contain compressed air at a pressure of 4 MN/m². The poisson's ratio is 0.3 and Young's modulus of cylindrical material is 210GN/m². The tangential stress in the cylinder material will be $\rightarrow 300$ MN/m²
57. A steel cylinder 20 m long having a mean diameter of 3 m and a wall thickness of 20 mm is to contain compressed air at a pressure of 4 MN/m². The poisson's ratio is 0.3 and Young's modulus of cylindrical material is 210GN/m². The change in length caused by the internal pressure will be $\rightarrow 5.7$ mm
58. A steel cylinder 20 m long having a mean diameter of 3 m and a wall thickness of 20 mm is to contain compressed air at a pressure of 4 MN/m². The poisson's ratio is 0.3 and Young's modulus of cylindrical material is 210GN/m². The change in diameter due to the internal pressure will be $\rightarrow 3.6$ mm
59. A thick cylinder designed as per the theory of thin cylinders, will be \rightarrow on unsafe side
60. A thick cylinder of internal radius R_i and the external radius R_o is subjected to internal fluid pressure p_i . Then the minimum radial stress is $\rightarrow 0$
61. A thick cylinder of internal radius R_i and the external radius R_o is subjected to internal fluid pressure p_i . Then the maximum radial stress is $\rightarrow p_i$
62. A thick cylinder of internal radius R_i and the external radius R_o is subjected to internal fluid pressure p_i . Then the maximum circumferential stress is \rightarrow
63. A thick cylinder of internal radius R_i and the external radius R_o is subjected to internal fluid pressure p_i . Then the minimum circumferential stress is \rightarrow
64. A thick pressure vessel is always used for the generation of steam, as it can withstand pressures. \rightarrow low
65. A thick spherical shell is subjected to an internal fluid pressure p . then the radial stress at any radius x is given by \rightarrow

66. A thick spherical shell is subjected to an internal fluid pressure p . then the circumferential stress at any radius x is given by :->
67. A thick spherical shell is subjected to an internal fluid pressure p . then the radial stress at any radius x is given by :->
68. A thin cylinder contains fluid at a pressure of 30 kg/cm^2 , the inside dia of the shell is 60 cm and the tensile stress in the material is to be limited to 900 kg/cm^2 . The shell must have minimum wall thickness of :-> 10 mm
69. A thin cylinder designed as per the theory of thick cylinders, will be :->on safer side
70. A thin cylinder is subjected to an external fluid pressure. The hoop stress is :->compressive
71. A thin cylinder is the one in which :->the variation in hoop stress along the thickness can be neglected
72. A thin cylinder of dia 100 mm and thickness 5 mm is subjected to an internal fluid pressure of 10 N/mm^2 . the hoop stress is :-> 100 N/mm^2
73. A thin cylinder pipe is tested for internal fluid pressure. If it has failed due to hoop tension mainly, the crack will be :->longitudinal
74. A thin cylindrical steel pressure vessel of dia 6 cm and wall thickness 3 mm is subjected to an internal fluid pressure of intensity ' p '. If the ultimate strength of steel is 3600 kg/cm^2 , the bursting pressure will be :-> 360 kg/cm^2
75. A truss is completely analysed, when :->the direct stresses in all the members are found
76. A two dimensional structure is one which :->two dimensions are very much larger than the third
77. A uniform beam of effective length L , fixed at one end and loaded uniformly, will have maximum deflection at :-> $3/8L$ from free end
78. A uniform beam of effective length L , fixed at one end and loaded at the centre, will have maximum deflection at :-> **$L/4$ from free end**
79. By assuming that all the forces acting on a pin connected truss are co-planar and act at joints only it can be expected that all the members will be subjected to :->direct stresses only
80. For a thin cylinder the ratio longitudinal stress/hoop stress is :-> $1/2$
81. For a thin longitudinal cylinder the ratio hoop stress/longitudinal stress is :-> 2
82. For analyzing pin jointed frames by the method of joints, the joints must be selected in succession such that : -
>only two unknown forces exist at that joint
83. For analyzing pin jointed frames by the method of joints, the number of equations of static equilibrium available is :->any number of members but only three members with unknown forces
84. Free body diagram is :->the diagram of the body or a part of the body in isolated equilibrium
85. Hoop stress is :->circumferential tensile stress
86. If a cantilever beam subjected to a hydrostatic load with zero intensity at the free end and W at the fixed end. The maximum deflection is :->
87. If a cylinder is made up of thin-walled cylindrical sheets or laminations, the hoop stress will be of :->the most **favourable distribution with more or less uniform distribution over the entire thickness of compound shell**
88. If a simply supported beam subjected to a triangularly distributed load with its apex of magnitude ' W ' at the mid span. The maximum deflection is :->
89. If M , I and R denote the bending moment, the moment of inertia and the radius of curvature of the beam, which will represent the rate of loading :->
90. If M , I and R denote the bending moment, the moment of inertia and the radius of curvature of the beam, which

will represent the bending moment :->

91. If M , I and R denote the bending moment, the moment of inertia and the radius of curvature of the beam, which

of the following equation is correct :->

92. If the deflection at the free end of a uniformly loaded cantilever beam is 15 mm and the slope of the deflection

curve at the free end is 0.02 radian, then the length of the beam is :->1 m

93. If the deflection at the free end of a uniformly loaded cantilever beam of length 1 m is equal to 7.5 mm, the

slope at the free end is :->0.02 radian

94. If the length of a simply supported beam carrying a point load at the centre is doubled, the deflection at the

centre will become :->eight times

95. If the width 'b' and depth 'd' of a simply supported beam loaded at the centre are interchanged, the deflection at

96. If two forces acting at a joint are not along the same straight line, then for the equilibrium of the joint : ->each

force must be zero

97. In a compound cylinder, after shrinking-on if the radius at the junction is to be r , the internal radius of the outer

cylinder should be less than by d_1 and the external radius of the inner cylinder is greater than by d_2 . The circumferential strain is :-> dr/r

98. In a compound cylinder, after shrinking-on if the radius at the junction is to be r , the internal radius of the outer

cylinder should be less than by d_1 and the external radius of the inner cylinder is greater than by d_2 . The circumferential strain at the junction of the inner cylinder :-> d_2/r

the centre will change in the ratio :-> $(d/b)^2$

99. In a compound cylinder, after shrinking-on if the radius at the junction is to be r , the internal radius of the outer

cylinder should be less than by d_1 and the external radius of the inner cylinder is greater than by d_2 . The circumferential strain at the junction of the outer cylinder :-> d_1/r

100. In a compound cylinder, after shrinking-on if the radius at the junction is to be 10 cm, the internal radius

of the outer cylinder should be less than by 0.02 cm and the external radius of the inner cylinder is greater than

by 0.04. The circumferential strain at the junction of the outer cylinder :->0.02

101. In a conical tank the maximum meridional stress occurs when the tank is :->three quarter full

102. In a conical tank, the maximum hoop stress occurs when the tank is :->quarter full

103. In a cylinder wound with wire under tension, the cylinder will be before the introduction of fluid, under

:->hoop compression

104. In a pin jointed frame it is sufficient if the forces in all the members meeting at a joint are : ->coplanar

and concurrent

105. In a pin jointed frame the members meeting at a joint must be so arranged that :->the axes of all the members are concurrent and coplanar

106. In a pin jointed plane frame all the loads are assumed to act :->in the plane of the frame

107. In a thick cylinder for internal fluid pressure, the hoop stress will be maximum at :->inner surface

108. In a thick cylinder the hoop stress, along the thickness :->is of parabolic variation

109. In a thick cylinder the variation in hoop stress along the thickness is :->

110. In a thin spherical shell of radius 'r', wall thickness 't' when subjected to an internal pressure 'p' the total

force normal to the diameter plane would be :-> $\pi r^2 p$

111. In case of compound cylinders, the radial stress in both the inner and outer cylinders at their junction is

:->same

112. In case of compound cylinders, the stress induced in the inner cylinder is :->compressive hoop

113. In case of compound cylinders, the stress induced in the outer cylinder is :->tensile hoop
114. In case of thin cylindrical shell with hemispherical ends :->the thickness of cylindrical shell is more **than that of spherical ends**
115. In case of thin walled cylinders the ratio of hoop strain to longitudinal strain is :-
>
116. In case of thin walled cylinders the ratio of hoop strain to volumetric strain is :-
>
117. In case of thin walled cylinders the ratio of hoop stress to radial stress is :->1/2
118. In case of thin walled cylinders the ratio of longitudinal strain to volumetric strain is :-
>
119. In case of thin walled cylinders the ratio of longitudinal stress to shear stress is :->4
120. In general in plane framed determinate structures in equilibrium, if 'm' is number of members 'j' is number of joints and 'r' is the number of reaction components, then :-> $m + r = 2j$
121. In a beam ABC, AB is of length 'l' and simply supported. The span BC is of length l/4 and overhang over support B. At the free end of span BC, a point load 'W' is acting. The slope at the support A of the overhang beam is :->
122. In the case of thick cylinders, the longitudinal stress :->is uniform throughout the thickness
123. In two forces acting at a joint are not acting along the same straight line, then for equilibrium of the joint
:->both forces must be zero
124. Lame's theory is associated with :->thick cylindrical shells
125. Slope at the end of a simply supported beam of span 'l' with u.d.l w/unit length over the entire span is :-
-
>
126. Struts are load carrying members of a frame which are subjected to :->axial compressive force
127. System A is a simply supported beam with a load P at mid span. System B is the same beam but the load is replaced by a udl of intensity P/L where 'L' is the span. The mid span deflection of system B will :->be
>be
less than that of system A at mid span
128. The analysis of a structure is :->calculating the magnitude and nature of various straining actions at **salient points of the structure**
129. The assumption in the theory of bending that the material is homogeneous and isotropic and has the same value of Young's modulus in tension and compression implies that :->the stresses are proportional strains at all fibres
130. The assumption that the cross sections plane before bending remain plane even after bending means :->the strains in the fibres are proportional to the distances from the neutral axis
131. The basic form of a pin jointed frame is a :->triangle
132. The circumferential joint of two shells is classified as :->lap joint
133. The deflection by moment area method is given by :-> $\Delta x = EI$
134. The design of thin cylindrical shells is based on :->hoop stress
135. The equations of compatibility are written based on :->the geometry of the deformed structure under **the action of several forces acting**
136. The expression at any section for a beam is equal to :->B.M at the section
137. The expression at any section for a beam is equal to :->load intensity at the **section**
138. The hoop stress in a riveted cylindrical shell of dia (d), thickness (t), efficiency of the joint is and subjected to an internal pressure (p) is :-> $pd / 2t$
139. The hoop stress in a thin cylindrical shell is :->circumferential tensile stress
140. The hoop stress in the case of a thin spherical shell will be :->
141. The initial tension in the wire wound on a thin cylinder, upon introduction of fluid under pressure, will :-
>increase
142. The longitudinal stress in a thin cylinder subjected to internal fluid pressure of 'p' will be :-> $s d / 4t$
143. The maximum deflection of a fixed beam carrying a central load W is :->
144. The product of EI is called :->flexural rigidity
145. The radial pressure on the outer circumference of a thick cylinder is equal to :->atmospheric pressure

146. The ratio of central deflection in a beam freely supported at both ends to that when the beam is fixed at both ends and subjected to a central load W in both the cases would be :->4
147. The ratio of hoops stress in cylinder to hoop stress in spherical shell is equal to :->2
148. The ratio of maximum deflection of a beam to its span is called :->modulus of rigidity of the beam
149. The ratio of the maximum deflection of a cantilever beam with an isolated load at its free end and with a u.d.l. over its entire span, is :->8/3
150. The safe working pressure for a spherical vessel 1.5 m dia and 1.5 cm wall thickness with limiting tensile stress of 450 kg/cm² would be :->18 kg/cm²
151. The statement that "the deflection at any point in a beam subjected to any load system is given by the partial derivative of the strain energy stored w.r.t. the load acting at that point in the direction in which the deflection is described" is known as :->the first theorem of Castigliano
152. The volumetric strain in a thin spherical shell will be :->
153. The volumetric strain in the case of a thin cylinder subjected to internal pressure s , is :->
154. Thick walled cylinders are supposed to resist the pressure above :->2500 kg/cm²
155. Ties are load carrying members of a frame which are subjected to :->axial tensile force
156. Two closed thin vessels, one cylindrical and the other spherical with equal internal dia and wall thickness are subjected to equal internal fluid pressure. The ration of hoop stresses in the cylindrical to that of spherical vessels is :->2
157. Two inclined struts connected by a hinge at the ends and supporting a load at the joint cannot be supported over two masonry walls because :->the walls cannot take up horizontal component since tension develops due to bending
158. What is the maximum shear stress induced in a thin cylindrical shell subjected to an internal fluid pressure p ? :-> $pD/8t$
159. When a hollow thick cylinder is shrunk on to another thick cylinder, the inner cylinder will be subjected to :->hoop compression with maximum at inner surface
160. When a thick cylinder is subjected to external fluid pressure only, hoop stress will be maximum at :->inner surface
161. When a thick cylinder is subjected to external pressure only, maximum radial pressure will occur at :->outer surface
162. When a thick cylinder is subjected to internal fluid pressure only, maximum radial pressure will occur at :->outer surface
163. When a thin cylindrical shell is subjected to an internal pressure, there will be :->an increase in **diameter and length of the shell**
164. When the compound cylinder is subjected to internal pressure, the stress induces is :->tensile hoop
165. When the water in a closed vessel freezes, the cylinder will first rupture :->longitudinally
166. Which of the following represents the shear force at a section of the beam :->

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